



## CARL R. DARNALL ARMY MEDICAL CENTER

### MARISSA CALDWELL MECHANICAL OPTION





### CARL R. DARNALL ARMY MEDICAL CENTER

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- BUILDING OVERVIEW
- THESIS GOALS
- DEPTH
- DEDICATED OUTDOOR AIR SYSTEM VARIABLE REFRIGERANT FLOW WATER SOURCE HEAT PUMP AIR QUALITY ANALYSIS ENERGY COMPARISON STRUCTURAL BREADTH
- CONSTRUCTION BREADTH
- CONCLUSION



#### **BUILDING OVERVIEW**

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RIA **BUILDING OVERVIEW** THESIS GOALS DEPTH DEDICATED OUTDOOR AIR SYSTEM VARIABLE REFRIGERANT FLOW WATER SOURCE HEAT PUMP AIR QUALITY ANALYSIS **ENERGY COMPARISON** STRUCTURAL BREADTH **CONSTRUCTION BREADTH** CONCLUSION

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- **OWNER: ARMY CORPS OF ENGINEERS**
- DEPARTMENT OF DEFENSE UFC STANDARD 4-510
- LOCATED ON FORT HOOD BASE, OUTSIDE OF KILLEEN, TX
- FLOOR AREA: 900,000 SQ FT
- REPLACING THE CURRENT CARL R. DARNALL ARMY MEDICAL CENTER ACROSS STREET





#### **BUILDING OVERVIEW**

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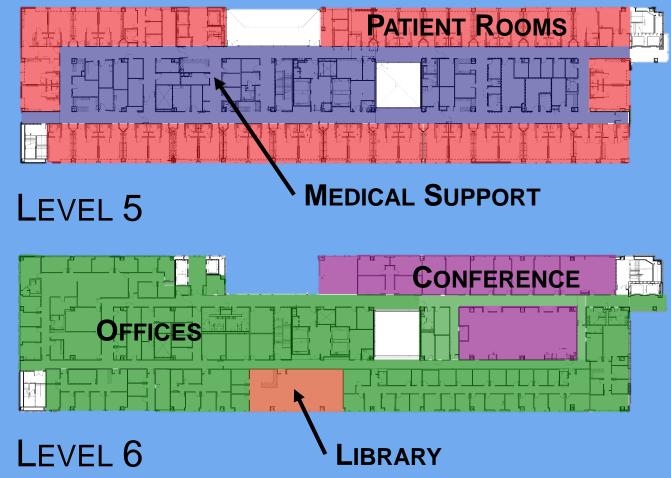
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#### BUILDING OVERVIEW

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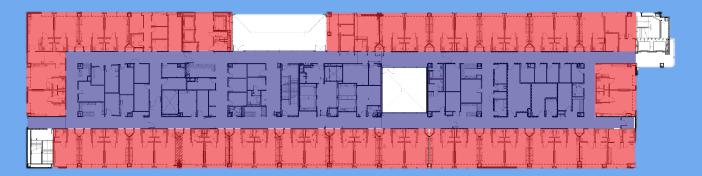
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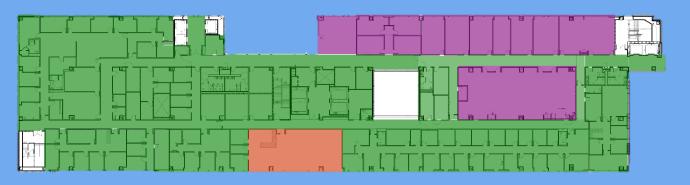
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LEVEL 5



LEVEL 6

#### MECHANICAL SYSTEM



(4) 1,300 TON CENTRIFUGAL CHILLERS (4) 18,000 MBH COOLING TOWERS (1) 200 TON HEAT RECOVERY CHILLER DOMESTIC HOT WATER & CHILLED WATER PROVIDES 44°F TO AHUS

RNA **BUILDING OVERVIEW** THESIS GOALS DEPTH DEDICATED OUTDOOR AIR SYSTEM VARIABLE REFRIGERANT FLOW WATER SOURCE HEAT PUMP AIR QUALITY ANALYSIS **ENERGY COMPARISON** STRUCTURAL BREADTH **CONSTRUCTION BREADTH** CONCLUSION

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#### COOLING SYSTEM

#### HEATING SYSTEM



(4) 11,700 MBH STEAM BOILERS LOW NOX BURNER & BOILER STACK ECONOMIZER (3) STEAM TO WATER HEAT EXCHANGER **STEAM FOR HUMIDIFICATION & STERILE EQUIPMENT** PROVIDES 140°F TO AHUS AND CAVS

#### MECHANICAL SYSTEM

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ENTHALPY WHEEL TO PRECONDITION OUTDOOR AIR (2) 45,000 CFM AIR HANDLER PROVIDING 55°F AIR CONSTANT AIR VOLUME TERMINAL UNITS WITH REHEAT

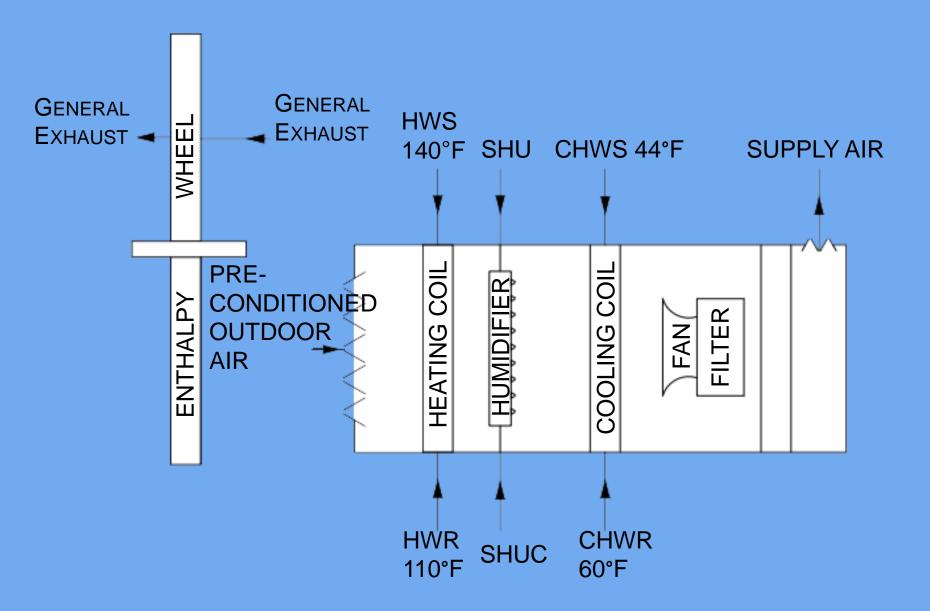
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#### AIRSIDE SYSTEM





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## **REDUCE ENERGY**



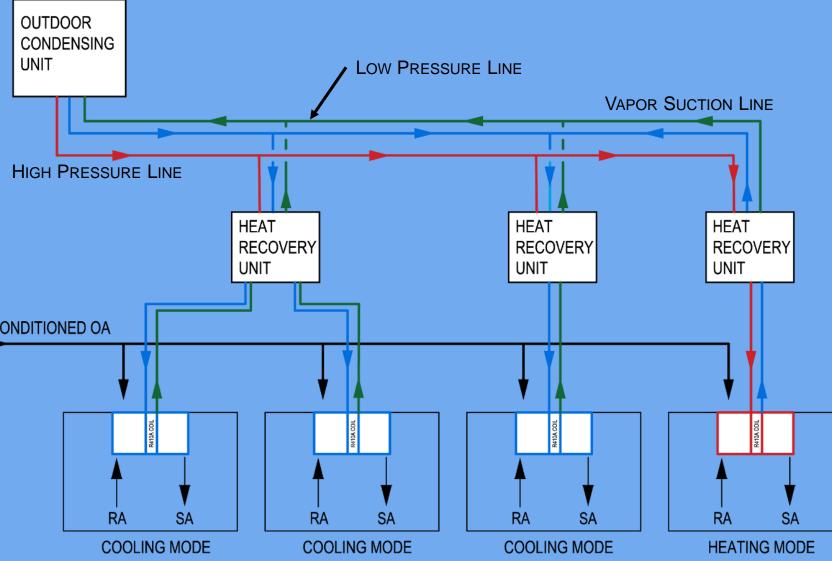
## SAFE COMMUNITY

## ECONOMICALLY FEASIBLE

## STRUCTURALLY STABLE







PRECONDITIONED OA

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RIA RMIT **BUILDING OVERVIEW** THESIS GOALS DEPTH DEDICATED OUTDOOR AIR SYSTEM VARIABLE REFRIGERANT FLOW WATER SOURCE HEAT PUMP AIR QUALITY ANALYSIS **ENERGY COMPARISON** STRUCTURAL BREADTH **CONSTRUCTION BREADTH** CONCLUSION

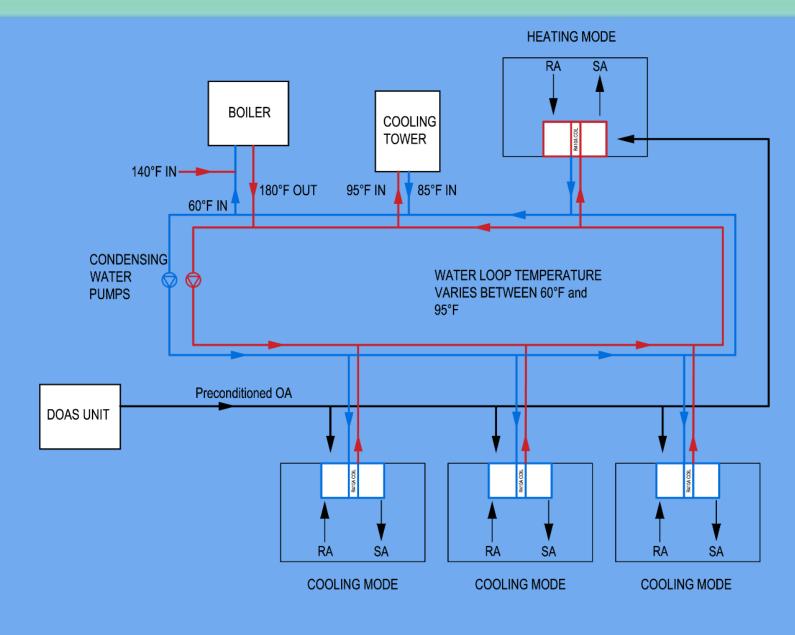
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### VARIABLE REFRIGERANT FLOW

### WATER SOURCE HEAT PUMP



### DEDICATED OUTDOOR AIR SYSTEM

400,000.0 중 350,000.0 300,000.0 250,000.0 **4** 200,000.0

150,000.0 **a** 100,000.0 50,000.0

0.0

DOAS DELIVERS AIR DIRECTLY TO THE SPACE IN VRF SYSTEM **ENTHALPY WHEEL WITH GENERAL EXHAUST** WSHP REDUCES BY 32%, VRF REDUCES BY 60%

ARNA **BUILDING OVERVIEW** THESIS GOALS DEPTH

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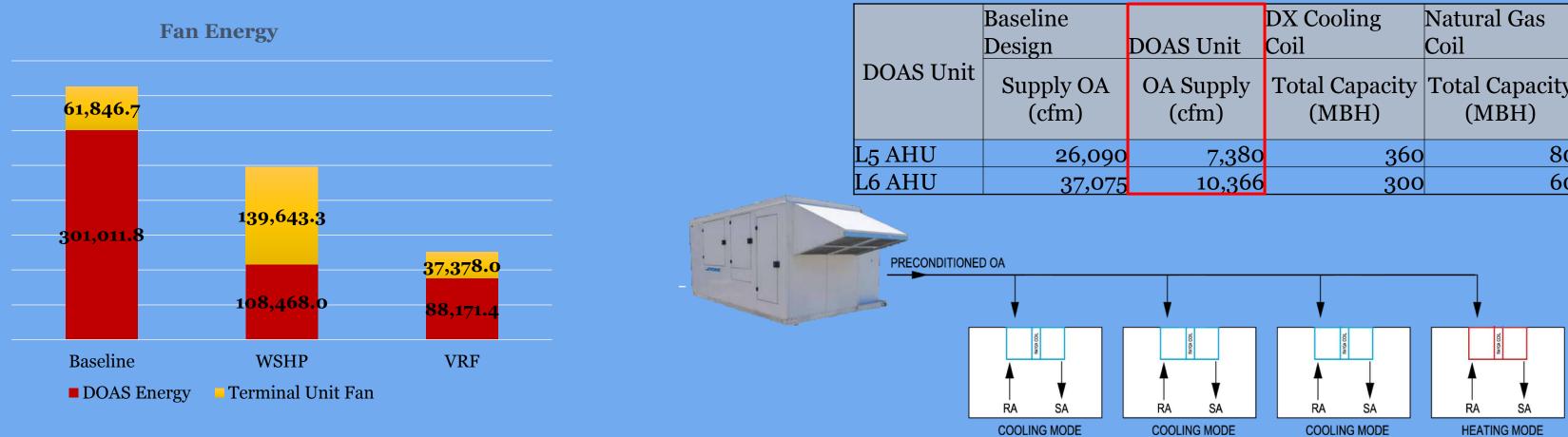
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DEDICATED OUTDOOR AIR SYSTEM VARIABLE REFRIGERANT FLOW WATER SOURCE HEAT PUMP AIR QUALITY ANALYSIS **ENERGY COMPARISON** STRUCTURAL BREADTH **CONSTRUCTION BREADTH** CONCLUSION

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	DX Cooling	Natural Gas
DOAS Unit	Coil	Coil
OA Supply (cfm)	Total Capacity (MBH)	Total Capacity (MBH)
7,380	360	80
10,366	300	60

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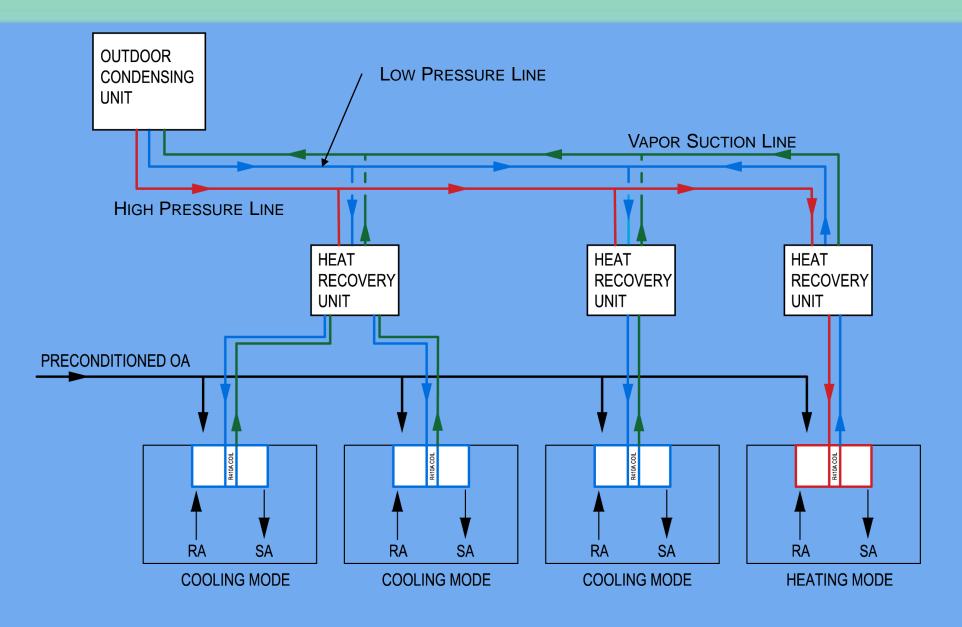
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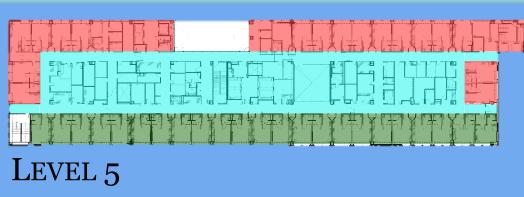
### VARIABLE REFRIGERANT FLOW

- VARIABLE REFRIGERANT FLOW WITH R-410A
- INDOOR VRF UNITS HAVE THE ABILITY TO OPERATE AS A CONDENSER AND A EVAPORATOR
- CONSISTS OF 3 PIPES: LOW PRESSURE, HIGH PRESSURE & VAPOR SUCTION LINE
- HEAT RECOVERY VS. HEAT PUMP
- DEFROST OPERATION



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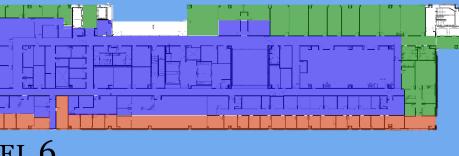
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## ASHRAE 15 COMPLIANCE



- INSTITUTIONAL OCCUPANCY & INDIRECT CLOSED SYSTEM
- CONCENTRATION MAY NOT EXCEED 50% LISTED IN TABLE 4-1
- MULTI-V REFRIGERANT CHARGE CALCULATOR
- SMALL ROOMS USE IN-DUCT VRF INDOOR UNITS

#### ASHRAE 15 REFRIGERANT CHARGE LIMIT



#### **REFRIGERANT CHARGE CALCULATIONS**

Zone Configuration	Zone	Outdoor Cooling	Unit Heating	Capacity (tons)	# Indoor Units	RCL (lb/Mcf)	Comply with ASHRAE 15?
Zoning 1	L5 South Exterior	264,000	297,000	22	23	22.14	NO
Zoning 2	L5 South East Ext	144,000	162,000	12	12	19.50	NO
Zonnig 2	L5 South West Ext	120,000	135,000	10	11	19.76	NO
	L5 South West Ext	96,000	108,000	8	11	12.17	YES
Zoning 3	L5 S Exterior Zone	96,000	108,000	8	10	12.15	YES
Zonnig 3	L5 SE Exterior Zone	96,000	108,000	8	10	11.82	YES
	L5 SE Corner Zone	96,000	108,000	8	10	11.68	YES

RCL			Highly Toxic or Toxic
ppm v/v	lb/Mcf	g/m^3	Under Code Classification
140,000	13	420	Neither

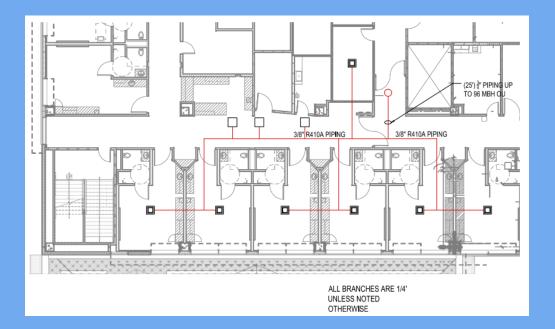
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INSTITUTIONAL OCCUPANCY & INDIRECT CLOSED SYSTEM CONCENTRATION MAY NOT EXCEED 50% LISTED IN TABLE 4-1 MULTI-V REFRIGERANT CHARGE CALCULATOR SMALL ROOMS USE IN-DUCT VRF INDOOR UNITS

## ASHRAE 15 COMPLIANCE



#### ASHRAE 15 REFRIGERANT CHARGE LIMIT

				RCL		Highly Toxic or Toxic
	OEL	Safety	nnm u/u	lb/Mof	$\sigma/m^{0}$	Under Code
	(ppm/v/v)	Group	ppm v/v	ppm v/v <mark>lb/Mcf</mark>		Classification
R-410A	1,000	A1	140,000	13	420	Neither

#### **REFRIGERANT CHARGE CALCULATIONS**

		Outdoor	Unit	Capacity	#	RCL	Comply with
Zone Configuration	Zone	Cooling	Heating	(tons)	Indoor Units	(lb/Mcf)	ASHRAE 15?
Zoning 1	L5 South Exterior	264,000	297,000	22	23	22.14	NO
Zoning 2	L5 South East Ext	144,000	162,000	12	12	19.50	NO
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	L5 South West Ext	96,000	108,000	8	11	12.17	YES
Zoning 3	L5 S Exterior Zone	96,000	108,000	8	10	12.15	YES
Zonnig 3	L5 SE Exterior Zone	96,000	108,000	8	10	11.82	YES
	L5 SE Corner Zone	96,000	108,000	8	10	11.68	YES

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 INTERIOR & EXTERIOR ROOMS PLACED ON SAME SYSTEM
ZONES BASED ON THE S EXTERIOR ZONE'S OUTDOOR UNIT'S CAPACITY
FOUR WAY CASSETTES FOR PATIENT ROOMS, IN-DUCT FOR OFFICES

#### ZONING



		Outdoor	Unit	Capacity	#	RCL	Comply with	
Zone Configuration	Zone	Cooling	Heating	(tons)	Indoor Units	(lb/Mcf)	ASHRAE 15?	
	L5 South Exterior	264,000	0	22		22.14	NO	
Zoning 2	L5 South East Ext	144,000	162,000	12	12	19.50	NO	
Zoming 2	L5 South West Ext	120,000	135,000	10	11	19.76	NO	
	L5 South West Ext	96,000	108,000	8	11	12.17	YES	
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Zonnig 3	L5 SE Exterior Zone	96,000	108,000	8	10	11.82	YES	
	L5 SE Corner Zone	96,000	108,000	8	10	11.68	YES	

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Zone L5 NW Exterior Zon L5 W Interior Zone 5 SW Exterior Zone 5 S Exterior Zone 5 SE Corner Zone 5 SE Exterior Zone L5 NE Exterior Zone L5 NE Interior Zone L6 NW Interior Zon L6 Interior Zone L6 E Interior Zone L6 SW Exterior Zon L6 S Exterior Zone L6 SE Exterior Zone L6 NE Exterior Zon L6 NW Exterior Zon

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#### SIZING

	Critical Load		0	utdoor Un	it	_		
					Derated	Number	Max	
	Load	MBH	Cooling		Heating	of Units	Number	
ne	Cooling	66,238		81,000	68,850	11	13	
	Cooling	55,708	72,000	81,000	68,850	13	13	
e	Cooling	76,585	96,000	108,000	91,800	10	16	
	Cooling	78,562	96,000	108,000	91,800	13	16	
	Cooling	87,183	96,000	108,000	91,800	13	13	
e	Cooling	77,507	96,000	108,000	91,800	10	16	
e	Cooling	51,079	72,000	81,000	68,850	13	13	
<u>,</u>	Heating	37,148	72,000	81,000	68,850	16	16	
e	Cooling	51,814	72,000	81,000	68,850	13	13	
	Cooling	52,251	72,000	81,000	68,850	12	13	
	Cooling	68,708	72,000	81,000	68,850	12	13	
e	Cooling	68,374	72,000	81,000	68,850	13	13	
	Cooling	45,069	72,000	81,000	68,850	13	13	
Ś	Cooling	55,377	72,000	81,000	68,850	13	13	
e	Cooling	82,378	96,000	108,000	91,800	14	16	
ne	Cooling	59,672	72,000	81,000	68,850	9	13	



**OVERALL 16 OUTDOOR CONDENSING UNITS** LG MULTI-V

## DERATED HEATING CAPACITY BASED ON 25°F WINTER CONDITIONS MAX NUMBER OF INDOOR UNITS FOR OUTDOOR UNIT DEPENDS ON

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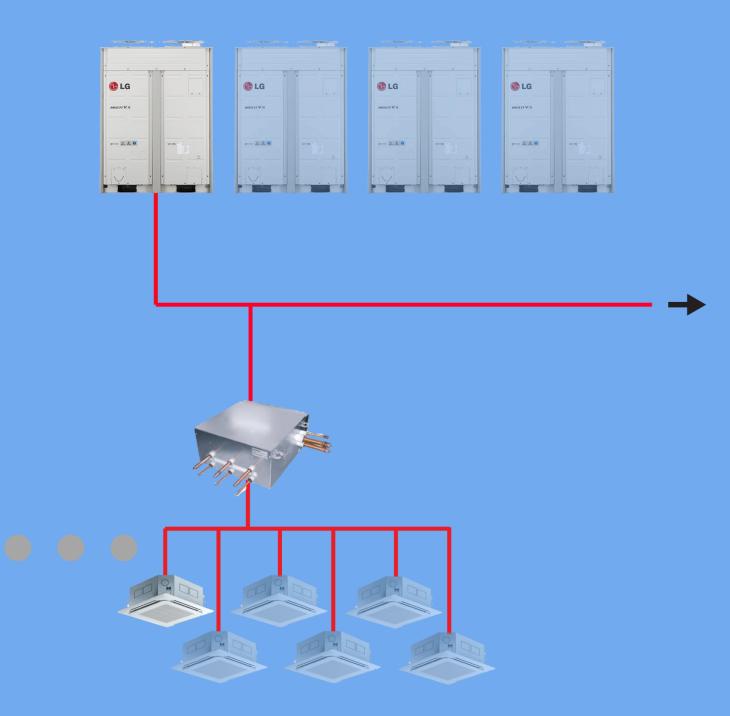
Room Name Medical Bed Medical Bed Medical Bed Medical Bed Medical Bed Medical Bed Pediatric Equipment Soiled Utility Nurse Team Center Remote Monitor Station Lab/SAT Poct

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#### SIZING

Sensible from	Room Load	Adjusted	Heating	VRF Inde	oor Unit	WSH	IP Unit
AHU (btu/h)	(btu/h)	(btu/h)	(htu/h)	Cooling	Heating	Cooling	Heating
3,169	10,528	7,359	7,270	9,600	10,900	9,700	12,000
2,868					10,900	9,700	12,000
2,868				9.600	10,900	9,700	12,000
2,868	10,528	7,659	7,270	9,600	10,900	9,700	12,000
2,868	10,528	7,659	7,270	9,600	10,900	9,700	12,000
3,169	10,528	7,359	7,270	9,600	10,900	9,700	12,000
1,036	2,649	1,613	39	7,500	8,500	8,000	9,800
216	135	135	0				
1,192	2,570	1,378	40	7 500	8 500	8 000	0.800
755	2,649	1,894	39	7,500	8,500	8,000	9,800
265	2,649	2,384	39				



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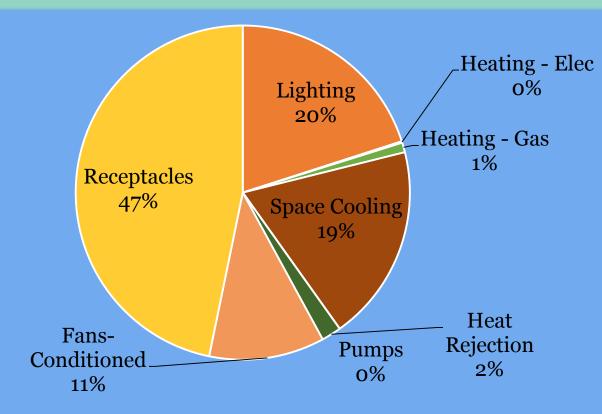
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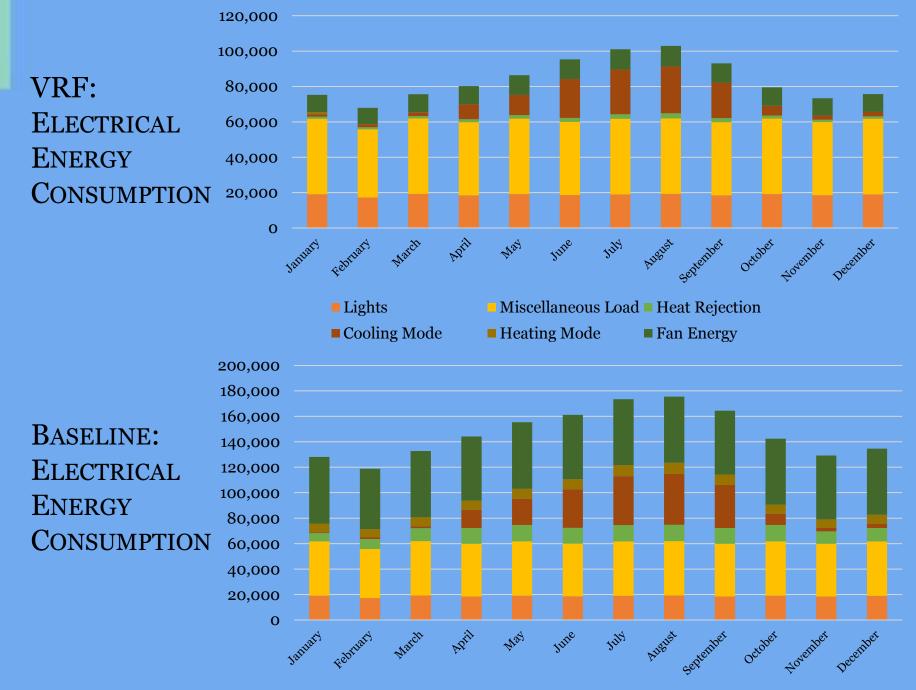
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**RECEPTACLE & LIGHTING DOMINATE ENERGY** SPACE COOLING INCLUDES BOTH COOLING & HEATING MODE HEAT REJECTION ENERGY OF THE CONDENSING UNIT VRF CONSUMES 753,443 KWH FEWER THAN BASELINE

## **ENERGY CONSUMPTION**





■ Lights ■ Miscellaneous Load ■ Heat Rejection ■ Chiller ■ Pump Energy ■ Fan Energy

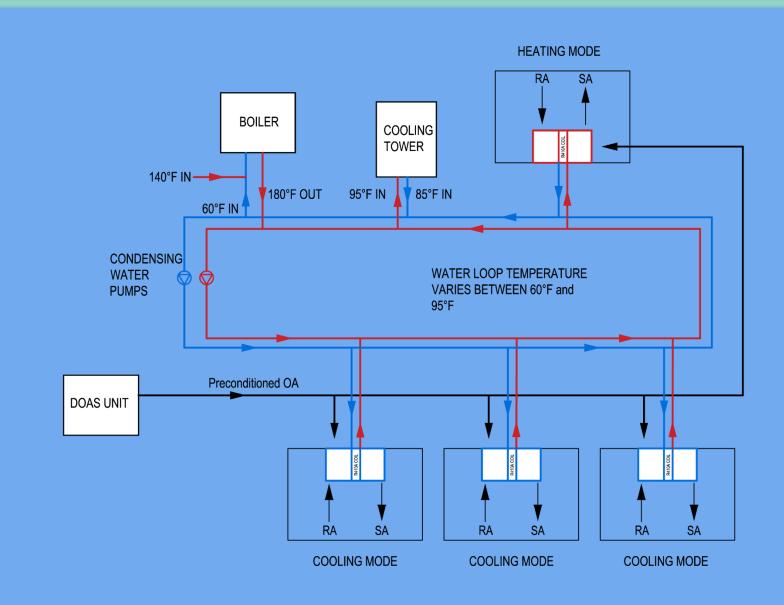
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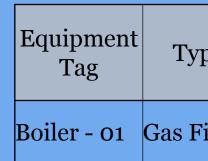
### WATER SOURCE HEAT PUMP

- WATER LOOP REMAINS BETWEEN 60°F & 95°F
- USES BOILER & COOLING TOWER WHEN IT CANNOT STAY IN THE TEMPERATURE RANGE
- Connects to existing hot water system for 140FMINIMUM FLOW
- ALL UNITS ON SAME LOOP, SHARE BETWEEN OFFICES & PATIENT ROOMS



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GAS FIRED MEETS PEA PRIMARY F ALL NEW F PL

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#### BOILER

/pe	Capacity (Btu/h)	Boiler Horse Power (BHP)	Fluid Flow (gpm)	Primary Fuel	NO <sub>X</sub> Emissions (ppm)
Fired	511,011	20.0	621.0	Natural Gas	40



#### GAS FIRED HOT WATER BOILER

- $MEETS \ PEAK \ HEATING \ LOAD \ DURING \ WINTER \ OPERATION$
- PRIMARY FUEL IS NATURAL GAS, SECONDARY FUEL IS FUEL OIL
- ALL NEW EQUIPMENT WILL BE LOCATED IN CENTRAL UTILITY PLANT

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Equipment Tag N 7 Cooling Tower -01

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MARLEY SOFTWARE USED FOR COOLING TOWER DESIGN FOR PEAK COOLING LOAD DURING SUMMER DESIGN ENTERING WATER TEMPERATURE OF 95°F WB

### COOLING TOWER



Design Nominal Tonnage	Motor Hp	Fluid Flow (gpm)	Design Ambient WB (°F)	Design EWT (°F)	Design LWT (°F)
228.0	7.5	636	78	95	85



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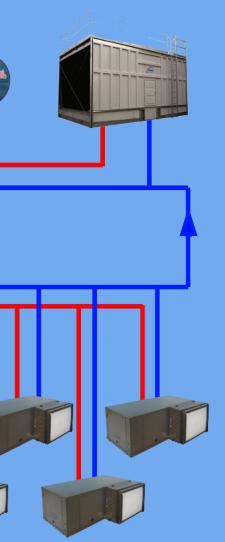
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### WATER SOURCE HEAT PUMP

Sensible from	Room Load	Adjusted	Heating	VRF Inde	oor Unit	WSH	IP Unit
AHU (btu/h)	(btu/h)	(btu/h)	(htu/h)		Heating	Cooling	Heating
3,169	10,528	7,359	7,270	9,600	10,900	9,700	12,000
2,868				-	10,900	9,700	12,000
2,868	10,528	7,659	7,270	9,600	10,900	9,700	12,000
2,868	10,528			9,600	10,900	9,700	12,000
2,868	10,528	7,659	7,270	9,600	10,900	9,700	12,000
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1,036	2,649	1,613	39	7,500	8,500	8,000	9,800
216	135	135	0				
1,192	2,570	1,378	40	7 500	8 500	8,000	9,800
755	2,649	1,894	39	7,500	8,500	0,000	9,000
265	2,649	2,384	39				



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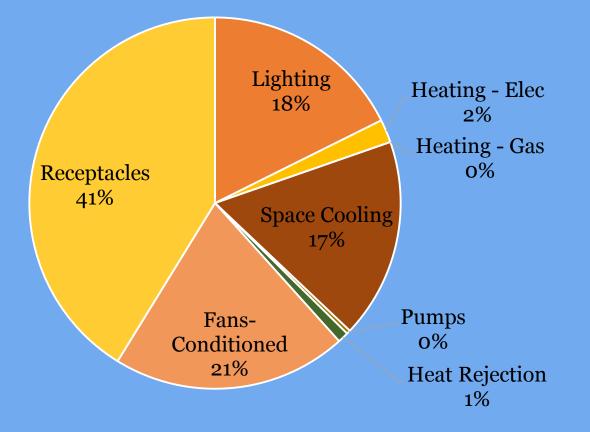
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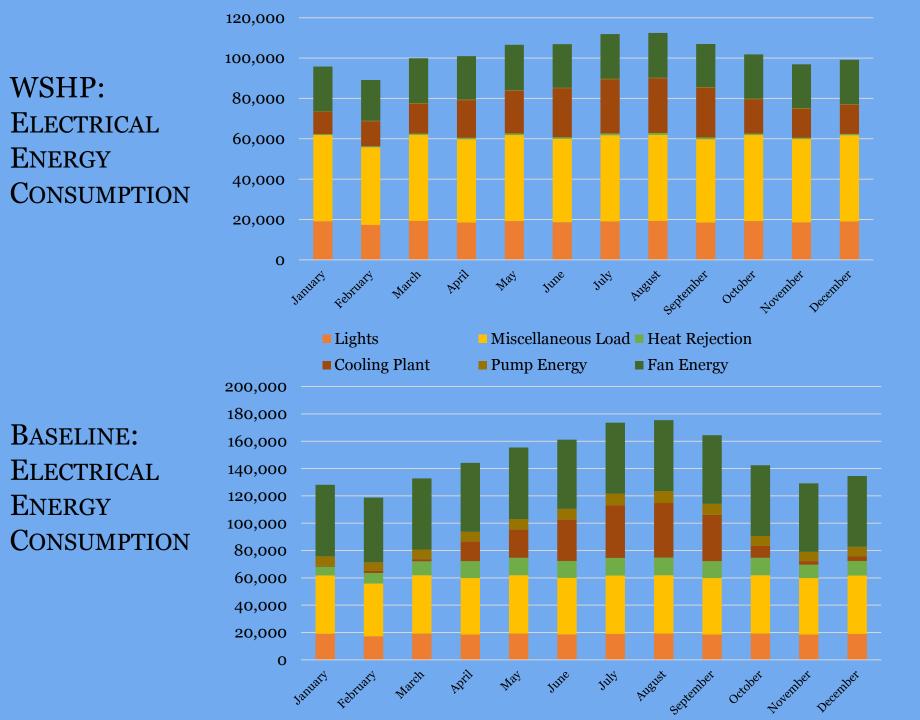
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**RECEPTACLE & FANS DOMINATE ENERGY CONSUMED** SPACE COOLING INCLUDES BOTH COOLING & HEATING MODE HEAT REJECTION ENERGY OF THE COOLING TOWER WSHP SAVES OVER 531,242 KWH THAN BASELINE

## **ENERGY CONSUMPTION**



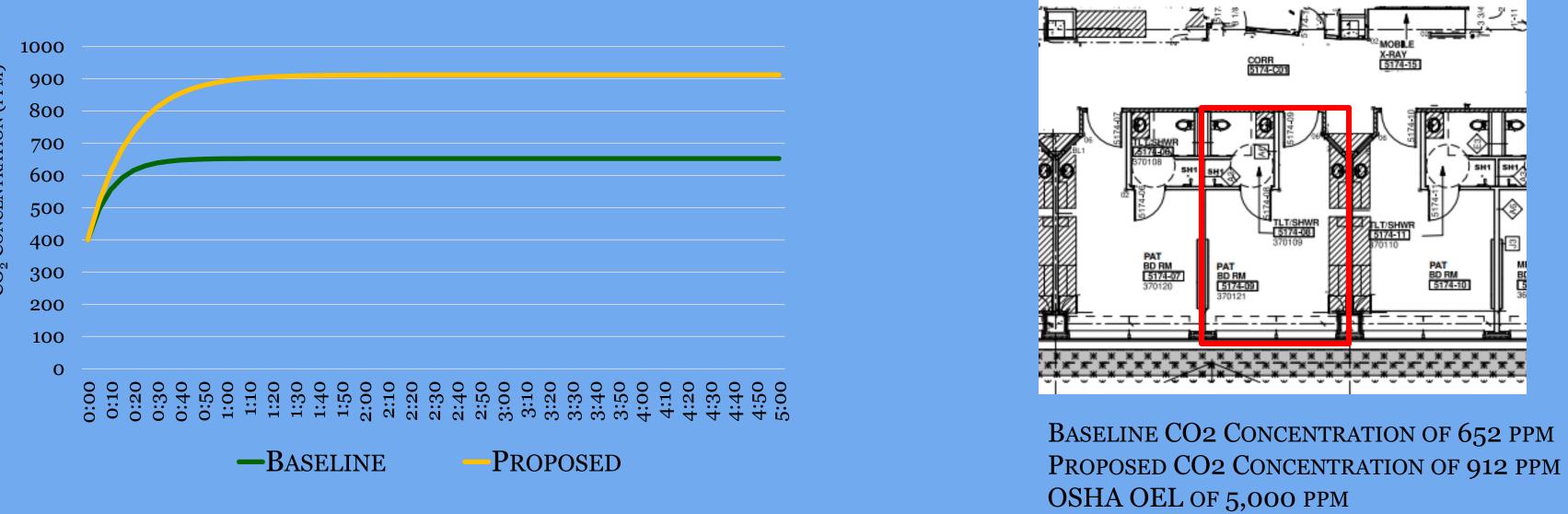


WSHP:

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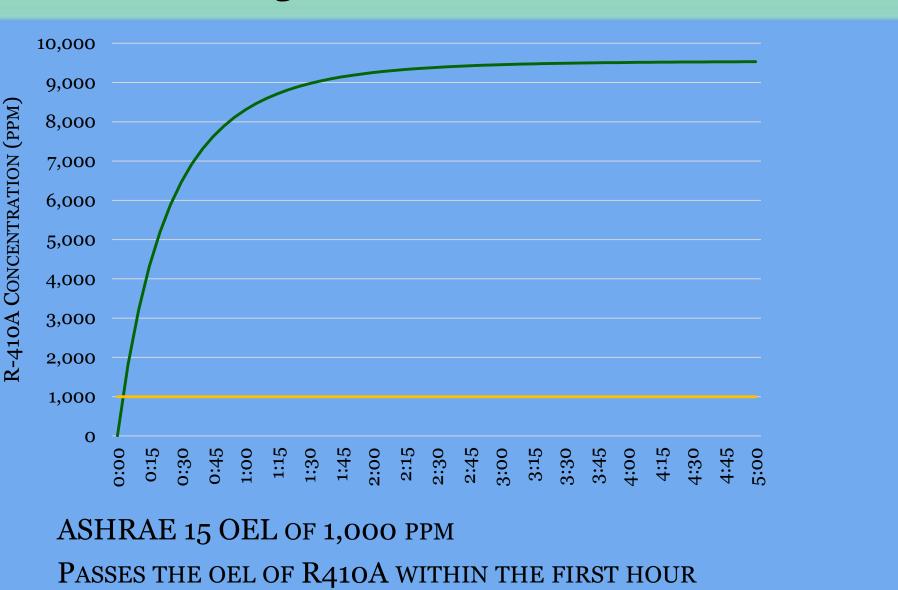
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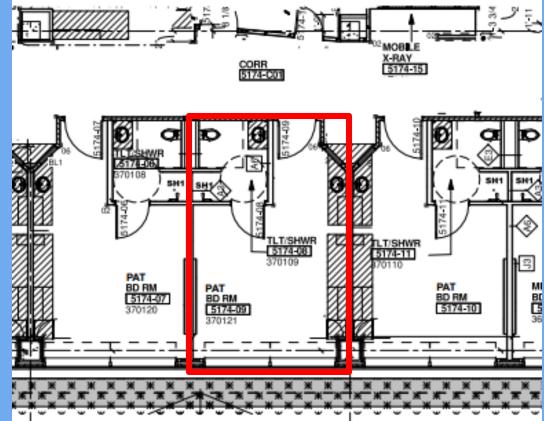
## AIR QUALITY ANALYSIS

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### AIR QUALITY ANALYSIS



#### REFRIGERANT LEAK BASED ON ASHRAE 15 APPENDIX B: TESTING LEAK AT RATE OF 2% BY MASS OF THE STARTING CHARGE PER HOUR

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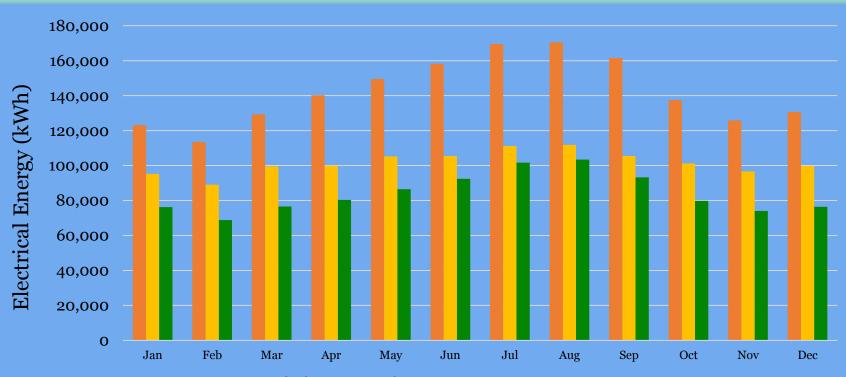
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### PAYBACK PERIOD

Initial Cost	VRF	WSHP
Expenses / Savings	Additional Costs	Initial Costs
Cooling Tower	-\$29,868	<b>\$0</b>
Boiler - 765 MBH	-\$17,800	<b>\$0</b>
Outdoor Condensing Unit	\$528,000	<b>\$0</b>
Chiller 270 ton	-\$190,500	-\$190,500
Terminal Units		\$285,600
Piping		\$426,920
DOAS Unit		-\$361,000
Costs	\$173,833	\$161,020
Maitnenance Costs		
16 Air Cooled Condensers	\$138,480	<b>\$0</b>
Cooling Tower	-\$8,872	<b>\$0</b>
Chiller - 270 ton	-\$104,744	-\$104,744
Boiler - 765 MBH		<b>\$0</b>
Maitnenance Costs	\$19,030	-\$104,744
Expenses		\$56,276
Annual Energy Costs		\$69,108
Simple Payback Period		0.81
Discounted Payback Period	8.72	2.96
Life Cycle Cost		
Savings		\$179,102.66
Cooling Tower Chiller - 270 ton Boiler - 765 MBH Maitnenance Costs Expenses Annual Energy Costs Simple Payback Period Discounted Payback Period Life Cycle Cost	-\$8,872 -\$104,744 -\$5,835 \$19,030 \$245,596 \$71,763 3.42 8.72 \$2,025,861.00	\$0 -\$104,74 \$0 -\$104,74 \$56,27 \$56,27 \$69,10 0.8 2.9 \$2,631,110.0



Original Design WSHP VRF LONGER PAYBACK PERIOD FOR VRF SYSTEM THAN WSHP SIMILAR INITIAL COSTS FOR BOTH **20** YEAR LIFE CYCLE

#### VARY WITHIN THE MAINTENANCE COSTS OF LARGER EQUIPMENT

## ENERGY & EMISSIONS COMPARISON

**BUILDING OVERVIEW** THESIS GOALS DEPTH DEDICATED OUTDOOR AIR SYSTEM VARIABLE REFRIGERANT FLOW WATER SOURCE HEAT PUMP AIR QUALITY ANALYSIS ENERGY COMPARISON STRUCTURAL BREADTH **CONSTRUCTION BREADTH** CONCLUSION

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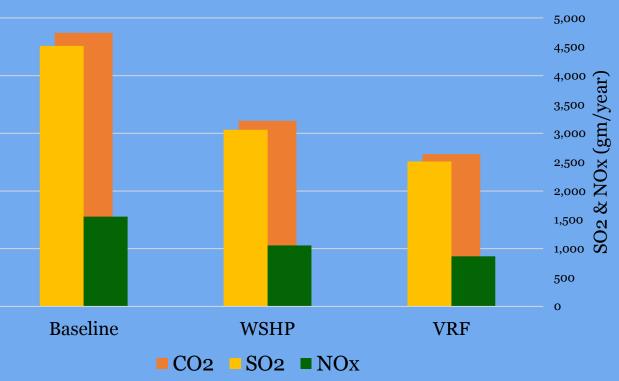
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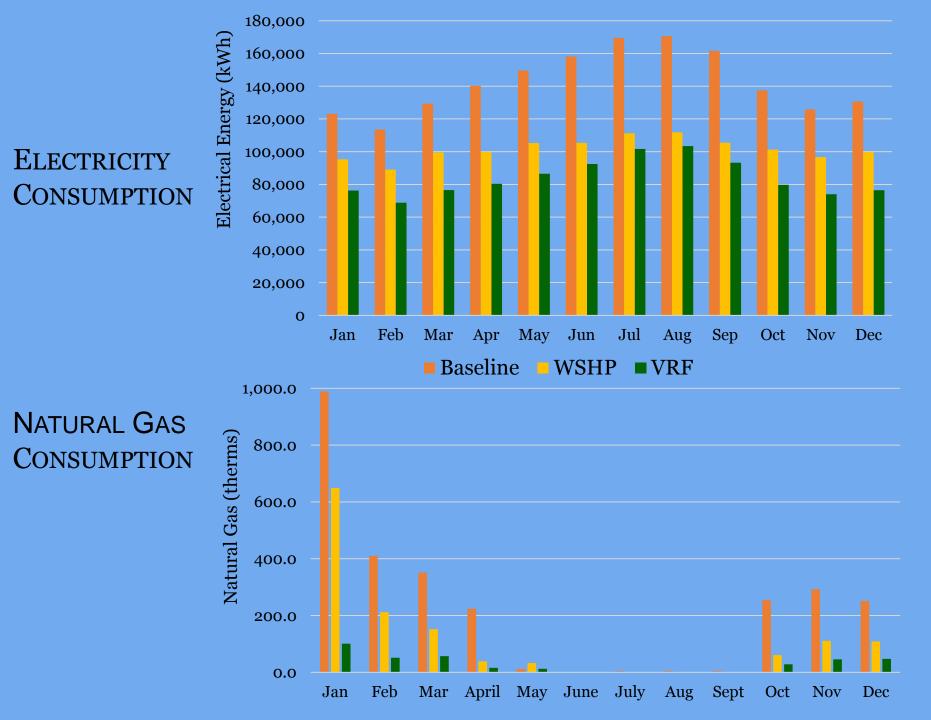
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LOW NOX P VRF ONI VSHP SA



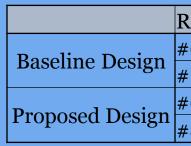
LOW NOX BURNER FOR BOILERS – 30 PPM, 40 PPM FOR PROPOSED

- VRF ONLY USES NATURAL GAS FOR DOAS UNIT
- WSHP SAVES 51% ON NATURAL GAS, VRF SAVES 72%



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Design

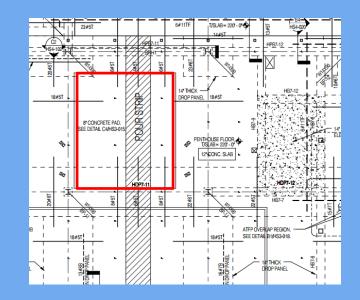
Baseline Reduced Loa

## STRUCTURAL BREADTH

Rebar Size	lb/ft	Number	Weight (lb)	Cost	Total	Savings
#6	1.50	24	1,219.9	\$1,205	¢ <del>с</del> с 40	\$4,248
#11	5.28	47	8,443.7	\$1,205 \$6,337	\$7,542	
#6	1.50	44	2,236.5	\$2,209	\$3,294	
#8	2.66	16	1,445.8	\$1,085	ə3,294	

	Column	Strip	Middle Strip		
	Neg. Moment	Pos. Moment	Neg. Moment	Pos. Moment	
	11 (#11)	18 (#11)	24 (#6)	18 (#11)	
d	16 (#7)	18 (#6)	12 (#6)	14 (#6)	

- COST SAVINGS OF \$4,248 FOR REDUCING LOAD ORIGINAL AHUS HAD 22,000 LB OPERATING WEIGHT
- DOWNSIZED STRUCTURAL SYSTEM BY ELIMINATING ANTI-FORCE PROTECTION & PROGRESS COLLAPSE RESISTANCE LOADS ANALYSIS COMPLIES WITH ACI 318 & DIRECT DESIGN METHOD



Equipment	Cooling Capacity (Btu/h)	Number	Operating Weight (lb)	
Outdoor Condensing Unit	72,000		4,972	
	96,000	5	2,865	
L5 DOAS Unit	360,000	1	4,284	
L6 DOAS Unit	300,000	1	4,282	

Material W6x20 Floor Decking (3" de 18 ga) Polypropylene Fiber Rebar Mat LW Concrete, 2-1/2'

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RIA **BUILDING OVERVIEW** THESIS GOALS DEPTH DEDICATED OUTDOOR AIR SYSTEM VARIABLE REFRIGERANT FLOW WATER SOURCE HEAT PUMP AIR QUALITY ANALYSIS **ENERGY COMPARISON** STRUCTURAL BREADTH **CONSTRUCTION BREADTH** CONCLUSION

CARL R.

ENTER

### **CONSTRUCTION BREADTH**

Removal of IBS Floor								
	Unit	Amount	Crew	Daily Output	Labor- Hours	Total Incl. O&P	Total	
	ft	4,148	8	600	0.093	43	\$178,364	
eep,	sq-ft	39,304	5	2850	0.011	4.29	\$168,614	
r	ft <sup>3</sup>	6,550	4	9500	0.004	1.07	\$7,009	
"	sq-ft	39,304	8	2585	0.022	3.3	\$129,703	
				Total		\$483,690		
				Entire B	uilding	\$483,690 \$6,046,125		

		Duration - Man Days					
Task	Baseline		VRF		WSHP		
TASK				Total	Man -		
	Man - Days	Total Days	Man - Days	Days	Days	Total Days	
FR&P Interstitial Floors	79.96	10	0.00	0	0.00	Ο	
Install Interstitial Floor							
Deck	65.36	8	0.00	0	0.00	Ο	
Install terminal units	51.00	26	76.98	38	76.98	38	
Piping	66.29	4	16.82	17	12.83	13	
Sheetmetal	32.00	11	21.33	11	21.33	11	
	Total	58	Total	66	Total	62	
		Difference	in Duration	-8		-4	

#### REMOVING IBS FLOORS GAINED 18 DAYS

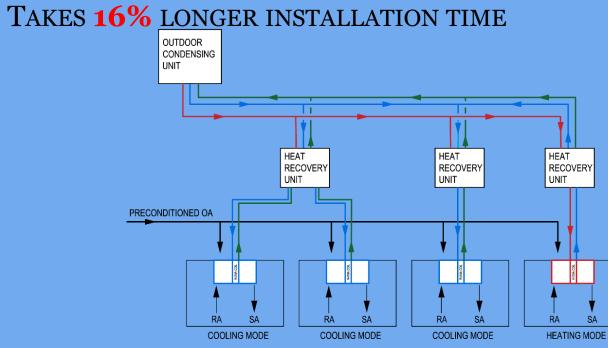
- AVERAGE INSTALLATION RATE OF TERMINAL UNITS IN A INTERSTITIAL FLOOR IS 30 MINUTES QUICKER
- PIPING INSTALLATION DOES NOT TAKE INTERSTITIAL FLOOR INTO ACCOUNT

CARL

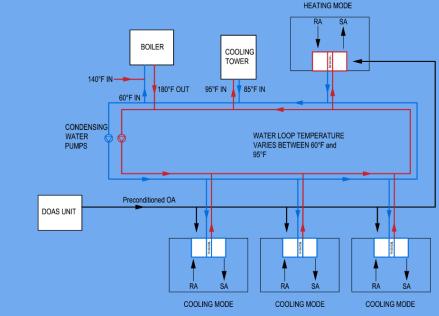
CONSUMPTION

### RECOMMENDATIONS

- VARIABLE REFRIGERANT FLOW
- SAVES **60%** IN ELECTRICAL ENERGY
- CONSUMES 72% LESS NATURAL GAS
- PAYBACK PERIOD OF 8.72 YEARS



SAVES **31%** IN ELECTRICAL ENERGY CONSUMPTION CONSUMES **51%** LESS NATURAL GAS PAYBACK PERIOD OF **2.96** YEARS TAKES 8% LONGER INSTALLATION TIME



#### WATER SOURCE HEAT PUMP



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CARL

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MECHANICAL DESIGN ENGINEER, MICHAEL MORDER SOUTHLAND INDUSTRIES PROJECT MANAGER, ASHLEY HAYES SOUTHLAND INDUSTRIES SHAWN MANLEY SENIOR MECHANICAL ENGINEER, SOUTHLAND INDUSTRIES THESIS ADVISOR, DR. W. BAHNFLETH PROFESSOR AT PENN STATE

PHOTOS COURTESY OF HKS INC. AND WINGLER & SHARP ARCHITECTS, INC.

### ACKNOWLEDGEMENTS



PENN STATE ARCHITECTURAL ENGINEERING FACULTY AE CLASS OF 2015 AE POWERPLAYERS





# QUESTIONS?

## CARL R. DARNALL ARMY MEDICAL CENTER REPLACEMENT

